THOMAS BERGER, ANNA-LENA RAUERT

Funnel cruise control

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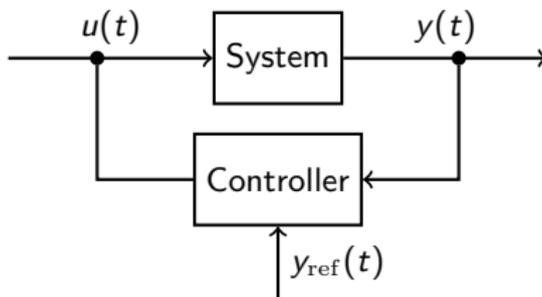
Control systems

$$\dot{x}(t) = f\big(x(t), u(t)\big), \quad x(0) = x^0$$
$$y(t) = h\big(x(t)\big)$$

- no knowledge of system parameters, only "structural assumptions" on the model of the system
- **Aim:** design simple controller such that $y(t)$ "tracks" $y_{\mathrm{ref}}(t)$

[SEIFRIED, BLAJER '13]

Angle: $0° < \alpha \leq 90°$

spring, damper with nonlinear characteristics: $K(z)$, $D(\dot{z})$

$u(t) = F$
$y(t) = x(t) + s(t)\cos\alpha$

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$$\begin{bmatrix} m_1 + m_2 & m_2\cos\alpha \\ m_2\cos\alpha & m_2 \end{bmatrix} \begin{pmatrix} \ddot{x} \\ \ddot{s} \end{pmatrix} = \begin{pmatrix} u \\ -K(s) - D(\dot{s}) + m_2 g\sin\alpha \end{pmatrix}$$

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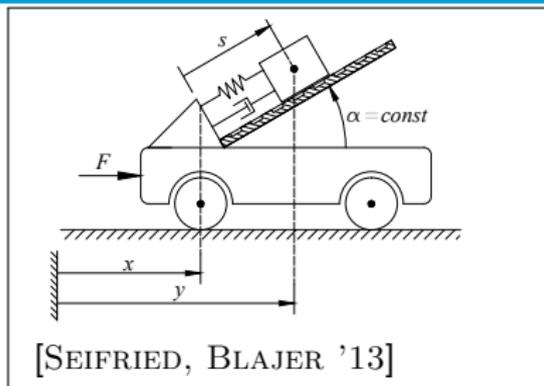
$$\begin{bmatrix} m_1 + m_2 & m_2\cos\alpha \\ m_2\cos\alpha & m_2 \end{bmatrix} \begin{pmatrix} \ddot{x} \\ \ddot{s} \end{pmatrix} = \begin{pmatrix} u \\ -K(s) - D(\dot{s}) + m_2 g \sin\alpha \end{pmatrix}$$

$$\dot{y} = \dot{x} + \dot{s}\cos\alpha$$

$$\ddot{y} = -c_1\big(K(s) + D(\dot{s}) - m_2 g \sin\alpha\big) + \frac{\sin^2\alpha}{m_1 + m_2\sin^2\alpha} u$$

$0° < \alpha \leq 90°$

$$\ddot{y} = f_1(s, \dot{s}) + \frac{\sin^2 \alpha}{m_1 + m_2 \sin^2 \alpha} u$$
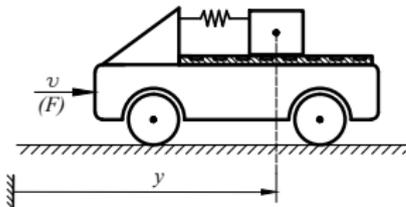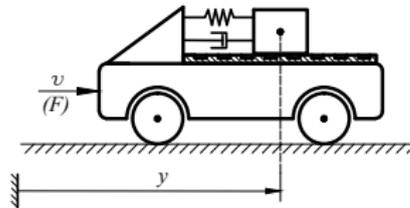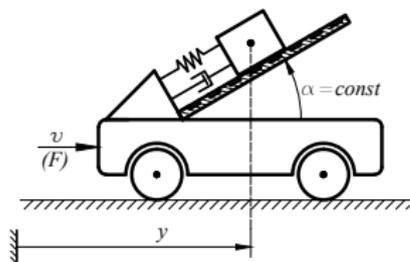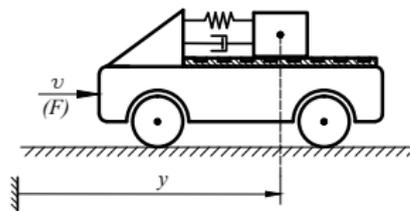
relative degree $= 2$

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$0° < \alpha \leq 90°$

$$\ddot{y} = f_1(s, \dot{s}) + \frac{\sin^2 \alpha}{m_1 + m_2 \sin^2 \alpha} u$$

relative degree $= 2$

$\alpha = 0°, \quad D'(\dot{s}) \neq 0$

$$y^{(3)} = f_2(s, \dot{s}) + \frac{D'(\dot{s})}{m_1 m_2} u$$

relative degree $= 3$

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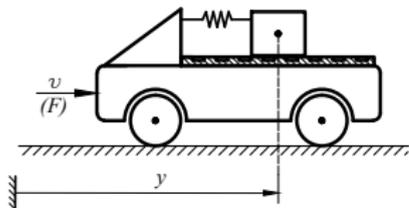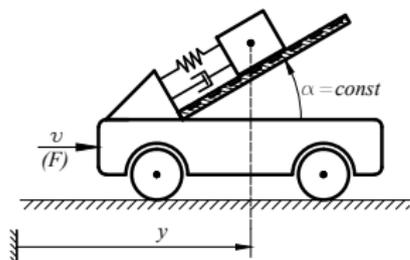
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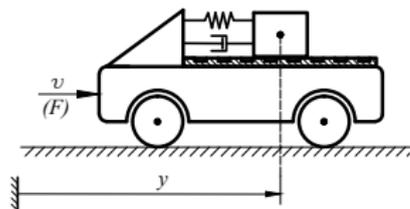
relative degree $= 2$


$\alpha = 0°, \quad D'(\dot{s}) \ne 0$

$$y^{(3)} = f_2(s, \dot{s}) + \frac{D'(\dot{s})}{m_1 m_2} u$$

relative degree $= 3$

$\alpha = 0°, \quad D'(\dot{s}) = 0, \quad K'(s) \ne 0$

$$y^{(4)} = f_3(s, \dot{s}) + \frac{K'(s)}{m_1 m_2} u$$

relative degree $= 4$
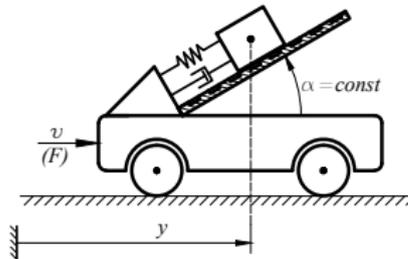
Internal dynamics: remaining dynamics when output is fixed
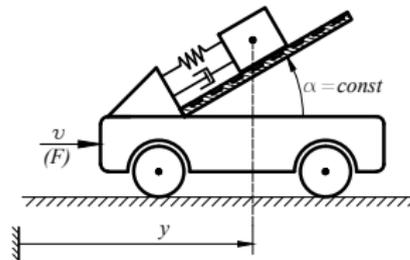
Internal dynamics: remaining dynamics when output is fixed

$$\ddot{\eta} = -c_3 K \left(\frac{\eta - y \cos \alpha}{\sin^2 \alpha} \right) - c_3 D \left(\frac{\dot{\eta} - \dot{y} \cos \alpha}{\sin^2 \alpha} \right) + c_4 g \sin \alpha$$
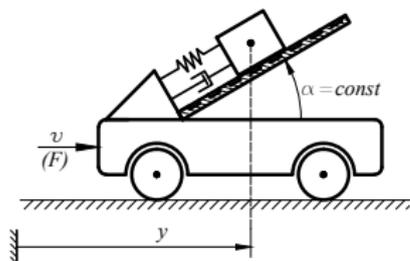
Internal dynamics: remaining dynamics when output is fixed
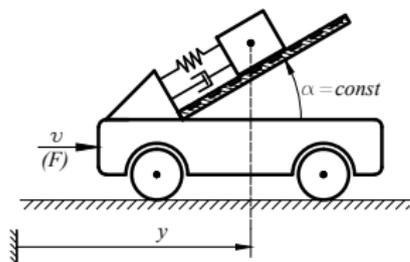
$$\alpha = 90°, m_2 = 1: \qquad \ddot{s} = -K(s) - D(\dot{s}) + g$$

Internal dynamics: remaining dynamics when output is fixed

$$\alpha = 90°, m_2 = 1: \qquad \ddot{s} = -K(s) - D(\dot{s}) + g$$

- Lyapunov function: kinetic + potential energy
- dissipativity: $D(\dot{s})\,\dot{s} \geq 0$

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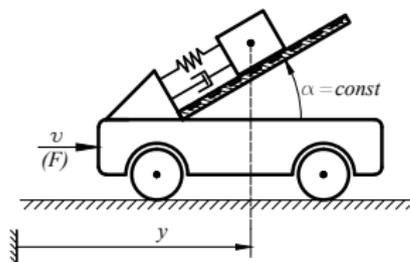
Internal dynamics: remaining dynamics when output is fixed

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- Lyapunov function: kinetic $+$ potential energy

- dissipativity: $D(\dot{s})\,\dot{s} \geq 0$

$$\implies s, \dot{s} \in L^\infty \quad \textbf{(stable internal dynamics)}$$

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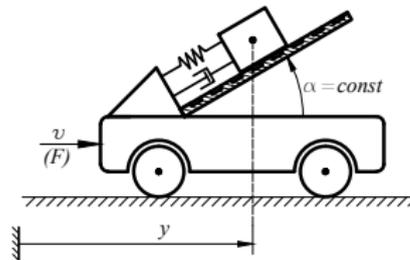
$0° < \alpha \leq 90°$

stable internal dynamics
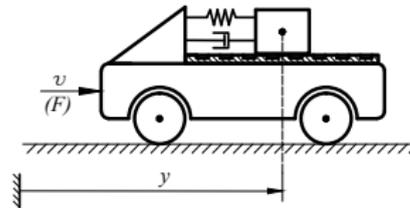
$\alpha = 0°, \quad D'(\dot{s}) \neq 0$

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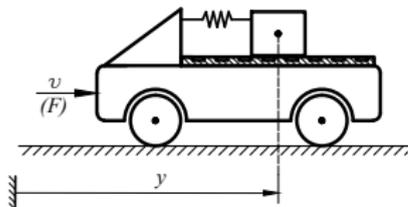
$\alpha = 0°, \quad D'(\dot{s}) = 0, \quad K'(s) \neq 0$

no internal dynamics
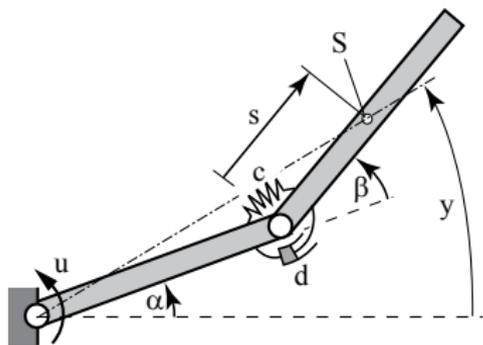
[SEIFRIED, BLAJER '13]

Rotational Manipulator Arm

Input: angular velocity of first link

Output: position of S described by angle y

relative degree $= 1$

unstable internal dynamics

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Reminder
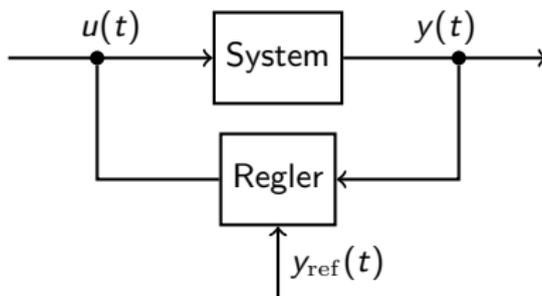
$$\dot{x}(t) = f\big(x(t), u(t)\big), \quad x(0) = x^0$$
$$y(t) = h\big(x(t)\big)$$

- no knowledge of system parameters, only: **known relative degree** and assumption of **stable internal dynamics**
- **Aim:** design simple controller such that $y(t)$ "tracks" $y_{\mathrm{ref}}(t)$

Funnel Control
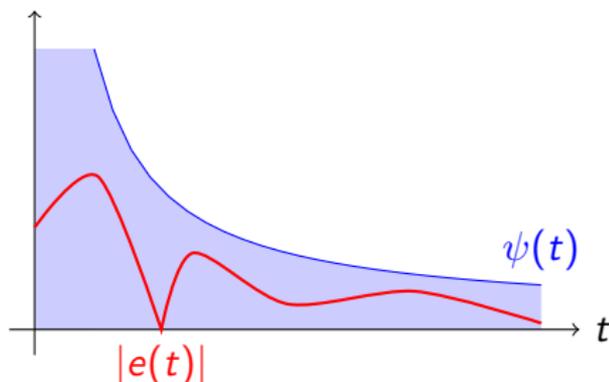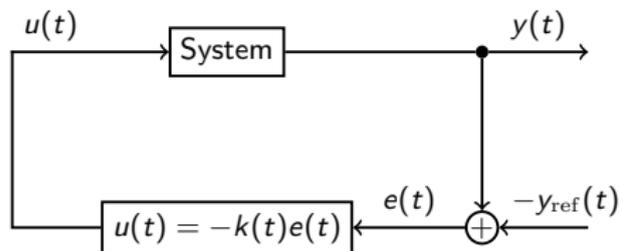
$$k(t) = \frac{1}{\psi(t) - |e(t)|}$$

[ILCHMANN, RYAN, SANGWIN '02]:
Works, if

- relative degree $= 1$
- stable internal dynamics

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Vehicle following framework
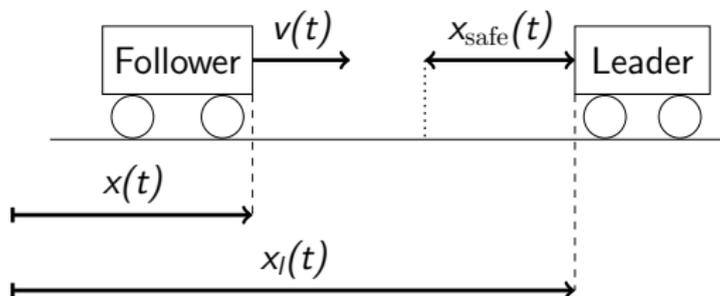
$$\dot{x}(t) = v(t), \quad m\dot{v}(t) = u(t) - F_g(t) - F_a\big(t, v(t)\big) - F_r\big(v(t)\big)$$

forces due to gravity: $\quad F_g(t) = mg \sin \theta(t)$

aerodynamic drag : $\quad F_a(t, v) = \frac{1}{2}\rho(t)C_d A v^2$

rolling friction : $\quad F_r(v) = mg C_r \mathrm{sgn}(v)$

Control objective
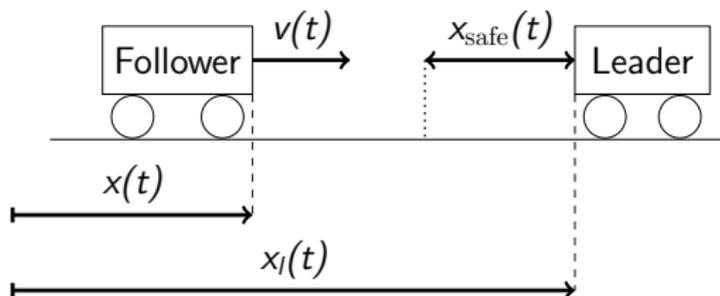
given: favourite speed $v_{\mathrm{ref}}(t)$, safety distance $x_{\mathrm{safe}}(t) = \lambda_1 v(t) + \lambda_2$

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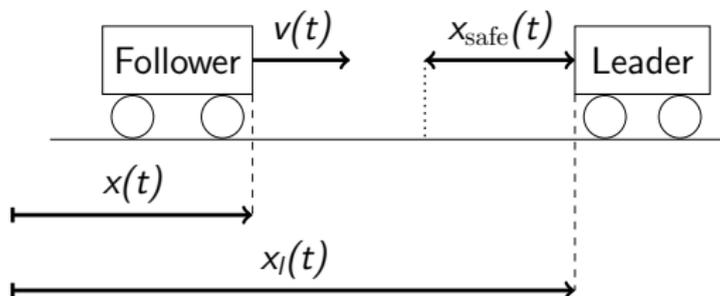
given: favourite speed $v_{\text{ref}}(t)$, safety distance $x_{\text{safe}}(t) = \lambda_1 v(t) + \lambda_2$

Aim: $u(t) = F(t, v(t), x_l(t) - x(t))$ s.t.

(O1) $x_l(t) - x(t) \geq x_{\text{safe}}(t)$

(O2) $|v(t) - v_{\text{ref}}(t)|$ is as small as possible such that (O1) is not violated

Funnel cruise control
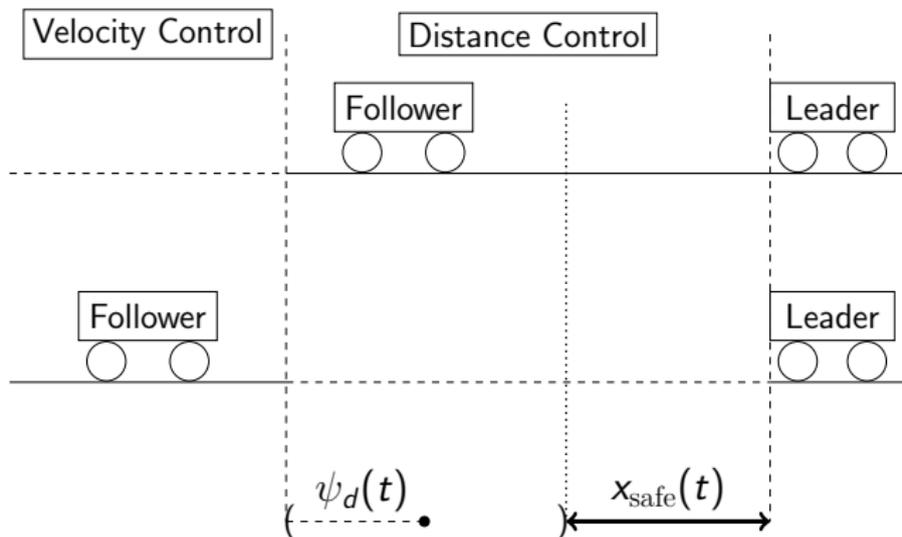
Velocity funnel control

tracking error: $\quad e_v(t) = v(t) - v_{\mathrm{ref}}(t) \in \mathcal{F}_v(t) := (-\psi_v(t), \psi_v(t))$

$\rightarrow$ treat $v(t)$ as output of the system

$$\dot{v}(t) = \tfrac{1}{m}u(t) + f_v\big(t, v(t)\big)$$

controller:

$$u_v(t) = -k_v(t)e_v(t),$$
$$k_v(t) = \frac{1}{\psi_v(t) - e_v(t)}$$

Distance funnel control

tracking error: $\qquad e_d(t) = x(t) - x_l(t) + x_{\mathrm{safe}}(t) + \psi_d(t)$

$$\in \mathcal{F}_d(t) := (-\psi_d(t), \psi_d(t))$$

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Distance funnel control

tracking error: $\quad e_d(t) = x(t) - x_l(t) + x_{\mathrm{safe}}(t) + \psi_d(t)$

$$\in \mathcal{F}_d(t) := (-\psi_d(t), \psi_d(t))$$

output: $\quad y(t) = \lambda_1 v(t) + x(t),$

reference signal: $\quad y_{\mathrm{ref}}(t) = x_l(t) - \lambda_2 - \psi_d(t)$

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output:
$$y(t) = \lambda_1 v(t) + x(t),$$

reference signal:
$$y_{\mathrm{ref}}(t) = x_l(t) - \lambda_2 - \psi_d(t)$$

$$\dot{x}(t) = v(t) = -\tfrac{1}{\lambda_1} x(t) + \tfrac{1}{\lambda_1} y(t),$$
$$\dot{y}(t) = \tfrac{\lambda_1}{m} u(t) + f_d\big(t, x(t), y(t)\big)$$

controller:
$$u_d(t) = -k_d(t) e_d(t),$$
$$k_d(t) = \frac{1}{\psi_d(t) - e_d(t)}$$

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Final control design

Intuitive approach:

- if $e_d(t) = -\psi_d(t)$,
  then switch from u_v to
  u_d and vice versa

- while u_d is active it
  should be possible that
  $e_v(t) < -\psi_v(t)$, but
  not $e_v(t) \geq \psi_v(t)$

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Problem:

- when $e_d(t) \to -\psi_d(t)$ we have $k_d(t) \nearrow \infty$
- when $e_v(t) \to -\psi_v(t)$ we have $k_v(t) \nearrow \infty$

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Problem:

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- when $e_v(t) \to -\psi_v(t)$ we have $k_v(t) \nearrow \infty$

Final control design

$$u(t) = \begin{cases} u_v(t), & e_d(t) \leq -\psi_d(t) \ \wedge \ e_v(t) \in \mathcal{F}_v(t), \\ u_d(t), & e_v(t) \leq -\psi_v(t) \ \wedge \ e_d(t) \in \mathcal{F}_d(t), \\ \min\{u_v(t), u_d(t)\}, & e_v(t) \in \mathcal{F}_v(t) \ \wedge \ e_d(t) \in \mathcal{F}_d(t) \end{cases}$$

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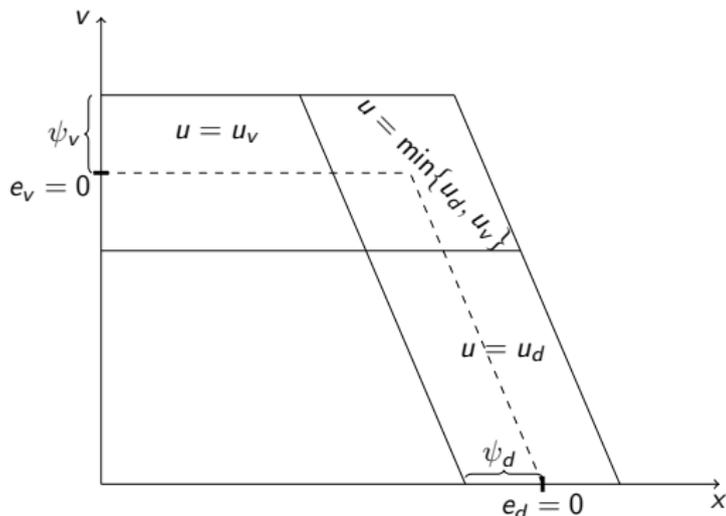
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Theorem [B., RAUERT '18]

$v_{\text{ref}}, x_l \in \mathcal{W}^{1,\infty}(\mathbb{R}_{\geq 0} \to \mathbb{R}), \ v_{\text{ref}} \geq 0, \ x_{\text{safe}} = \lambda_1 v(t) + \lambda_2$
$\implies x, v, u \in L^\infty$ and

$$\psi_v(t) - e_v(t) \geq \varepsilon, \quad \psi_d(t) - e_d(t) \geq \varepsilon,$$
$$\max\{0, \psi_v(t) + e_v(t)\} + \max\{0, \psi_d(t) + e_d(t)\} \geq \varepsilon$$

Sketch of the Proof

- existence of a local solution of the closed-loop system on $[0, \omega)$ (continuity of u)

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- existence of a local solution of the closed-loop system on $[0, \omega)$ (continuity of u)
- boundedness of x and v on $[0, \omega)$ using the sets
  $M_d = \{t \mid u(t) = u_d(t)\}$ and $M_v = \{t \mid u(t) = u_v(t)\}$

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Sketch of the Proof

- existence of a local solution of the closed-loop system on $[0, \omega)$ (continuity of u)
- boundedness of x and v on $[0, \omega)$ using the sets
  $M_d = \{t \mid u(t) = u_d(t)\}$ and $M_v = \{t \mid u(t) = u_v(t)\}$
- show $\psi_v(t) - e_v(t) \geq \varepsilon$ using that $u(t) \leq u_v(t)$ when $e_v(t) \geq 0$
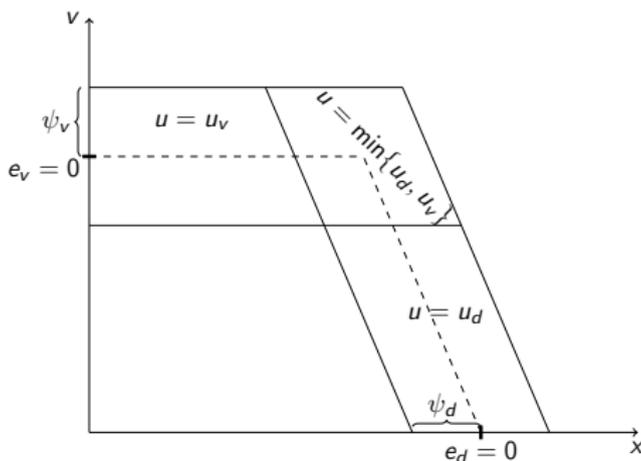
Sketch of the Proof - Part 2

- show $\psi_d(t) - e_d(t) \geq \varepsilon$ using that $u(t) \leq u_d(t)$ when $e_d(t) \geq 0$

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- show $\psi_d(t) - e_d(t) \geq \varepsilon$ using that $u(t) \leq u_d(t)$ when $e_d(t) \geq 0$
- show $\max\{0, \psi_v(t) + e_v(t)\} + \max\{0, \psi_d(t) + e_d(t)\} \geq \varepsilon$ by using that if $e_d(t^*) = -\psi_d(t^*)$, then $u(t) = u_v(t)$ in a neighborhood of t^*; if $e_v(t^*) = -\psi_v(t^*)$, then $u(t) = u_d(t)$ in a neighborhood of t^*

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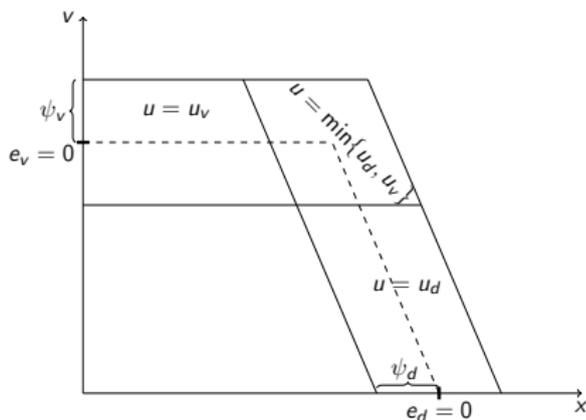
Sketch of the Proof - Part 2

- show $\psi_d(t) - e_d(t) \geq \varepsilon$ using that $u(t) \leq u_d(t)$ when $e_d(t) \geq 0$
- show $\max\{0, \psi_v(t) + e_v(t)\} + \max\{0, \psi_d(t) + e_d(t)\} \geq \varepsilon$ by using that if $e_d(t^*) = -\psi_d(t^*)$, then $u(t) = u_v(t)$ in a neighborhood of t^*; if $e_v(t^*) = -\psi_v(t^*)$, then $u(t) = u_d(t)$ in a neighborhood of t^*

- boundedness of u and $\omega = \infty$

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Simulation

m	$\theta(t)$	$\rho(t)$	C_d	C_r	A
1300 kg	0 rad	1.3 kg/m^3	0.32	0.01	2.4 m^2

x^0	v^0	λ_1	λ_2	v_{ref}
0 m	15 m s^{-1}	0.5 s	2 m	36 m s^{-1}

$$\psi_v(t) = 22.5e^{-0.2t} + 0.2, \quad \psi_d(t) = 4.$$

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